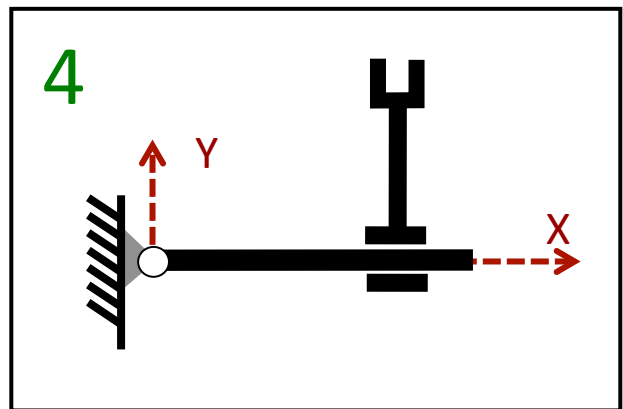
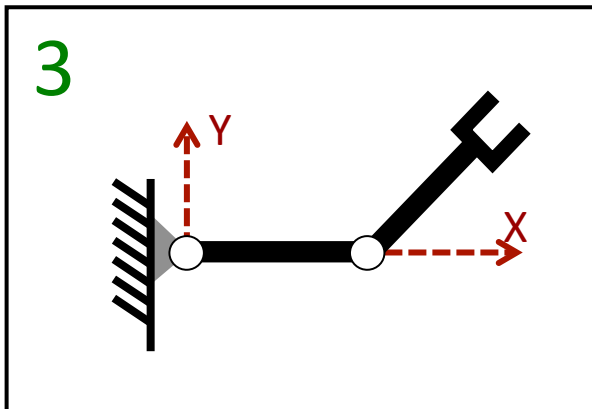
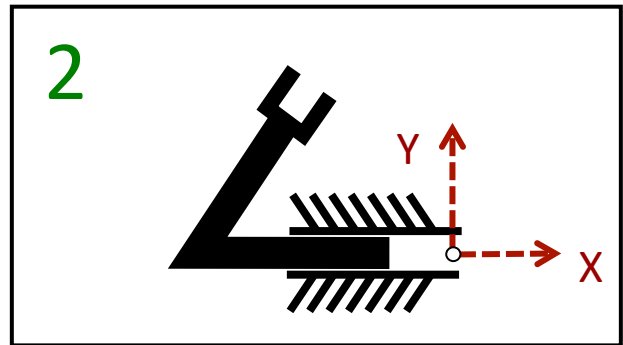
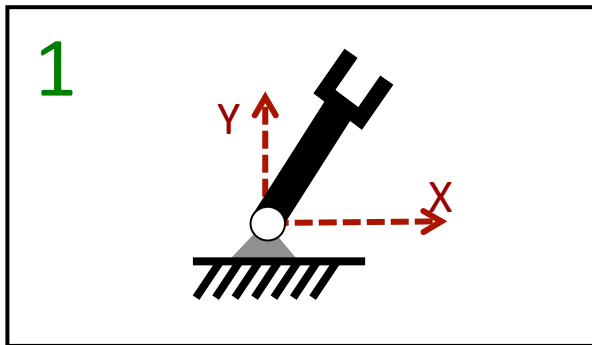


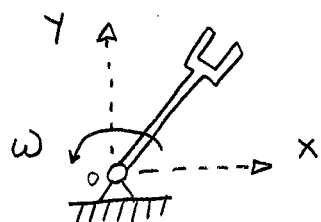
Exercise

Provide the spaces of twists of freedom and wrenches of constraint of these robots. I.e., give a basis of such spaces in the indicated reference frames.



OBSERVATION: Note that the technique used to compute W and Π is valid for any serial manipulator

Robot 1



In the shown reference frame, the twist of the end effector has the form

$$\hat{T}_e = \omega \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \quad (\text{in axis coords.})$$

↑
scalar
(rad/s)

Therefore, Π is the vector space spanned by $\begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$:

$$\Pi = \left\langle \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \right\rangle$$

↖ means "spanned by"

Since W is its reciprocal complement, it must be

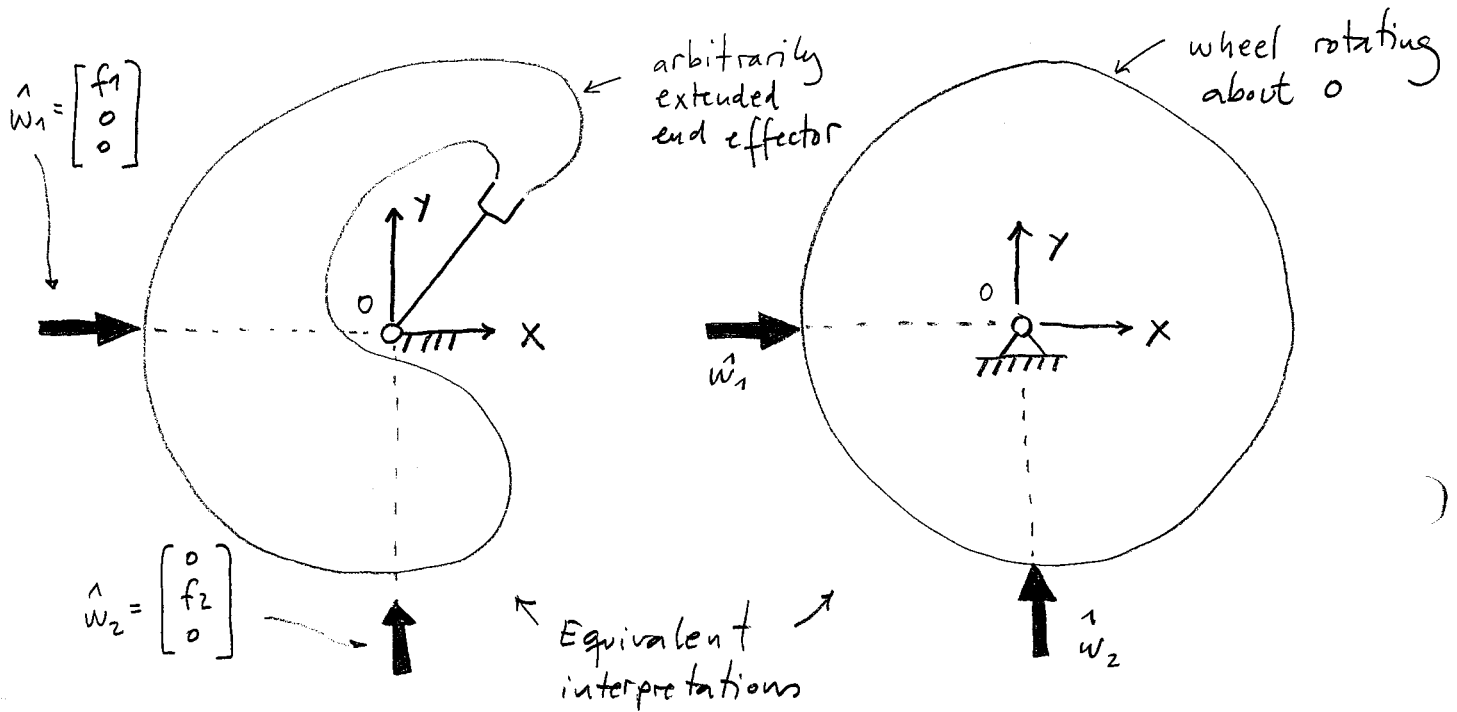
$$W = \left\langle \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}, \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} \right\rangle \quad (\text{in ray coords})$$

Interpretation: Since $W = \left\langle \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}, \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} \right\rangle$, the wrenches of constraint take the parametric form

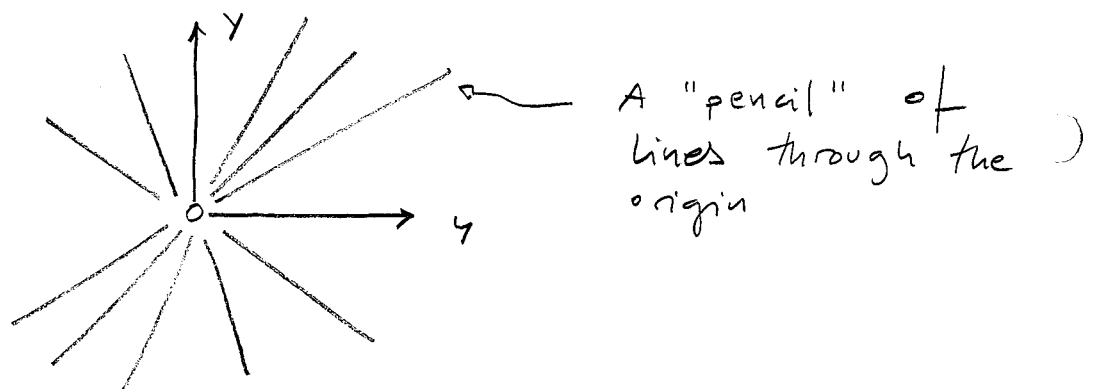
$$\hat{W}_e = \overset{\text{Newton}}{\leftarrow} f_1 \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} + \overset{\text{Newton}}{\rightarrow} f_2 \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 \\ 0 \end{bmatrix} \quad (\text{ray coords})$$

The wrenches of the form $\begin{bmatrix} f_1 \\ f_2 \\ 0 \end{bmatrix}$ have their line of action through the origin. A direction vector of the line is given by $\begin{bmatrix} f_1 \\ f_2 \end{bmatrix}$. These are the forces that the robot can equilibrate without activating its joints.

Graphically, we imagine the end effector as a lamina of arbitrary extension. The structurally supportable forces are the linear combinations of \hat{w}_1 and \hat{w}_2 :

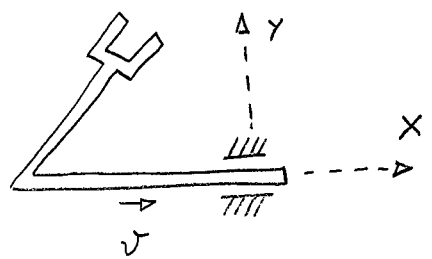


The linear combinations of \hat{w}_1 and \hat{w}_2 span a 2-system of wrenches, whose lines of action go through the origin:



This result could have been anticipated by geometric analysis. Note that since it must be $\hat{w}_e^T \cdot \hat{T}_e = 0 \neq \hat{T}_e \in \Pi$, $\hat{w}_e \in \mathcal{W}$, and $\hat{w}_e^T \cdot \hat{T}_e = f \cdot w \cdot \hat{S}_e^T \cdot \hat{S}_e$, the determination of \mathcal{W} reduces to finding the lines \hat{S}_e whose mutual moment with respect to \hat{S}_e is null. Since in this case $\hat{T}_e = w \begin{bmatrix} 0 \\ 1 \end{bmatrix} \Rightarrow \hat{S}_e = \begin{bmatrix} 0 \\ 1 \end{bmatrix} \Rightarrow$ the only lines with a null mutual moment relative to $\begin{bmatrix} 0 \\ 1 \end{bmatrix}$ are those of the previous pencil.

Robot 2



In the indicated frame, the end-effector twists take the form

$${}^1T_c = v \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

Thus:

$$\pi = \left\langle \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \right\rangle \Rightarrow W = \left\langle \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix}, \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \right\rangle$$

↙ axis
↖ ray

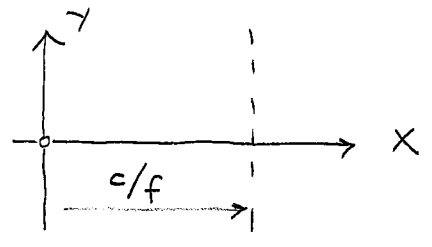
and the wrenches of constraint take the form:

$$\hat{w}_e = f \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} + c \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} = \begin{bmatrix} 0 \\ f \\ c \end{bmatrix}$$

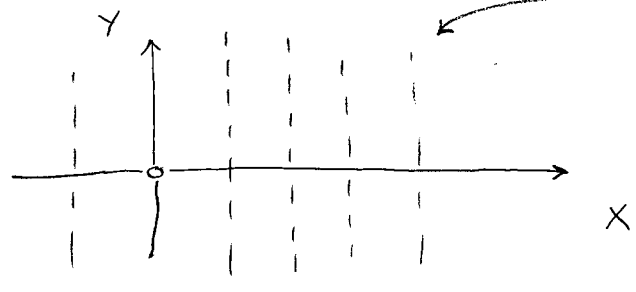
↙ Newton
↘ Newton · meter

where f and c are scalar signed magnitudes with force and moment units respectively.

The coordinates $\begin{bmatrix} f \\ c \end{bmatrix}$ describe a line parallel to the y axis at a distance c/f from the origin:



Thus, the structurally-supportable forces are those whose line of action belongs to the line bundle:

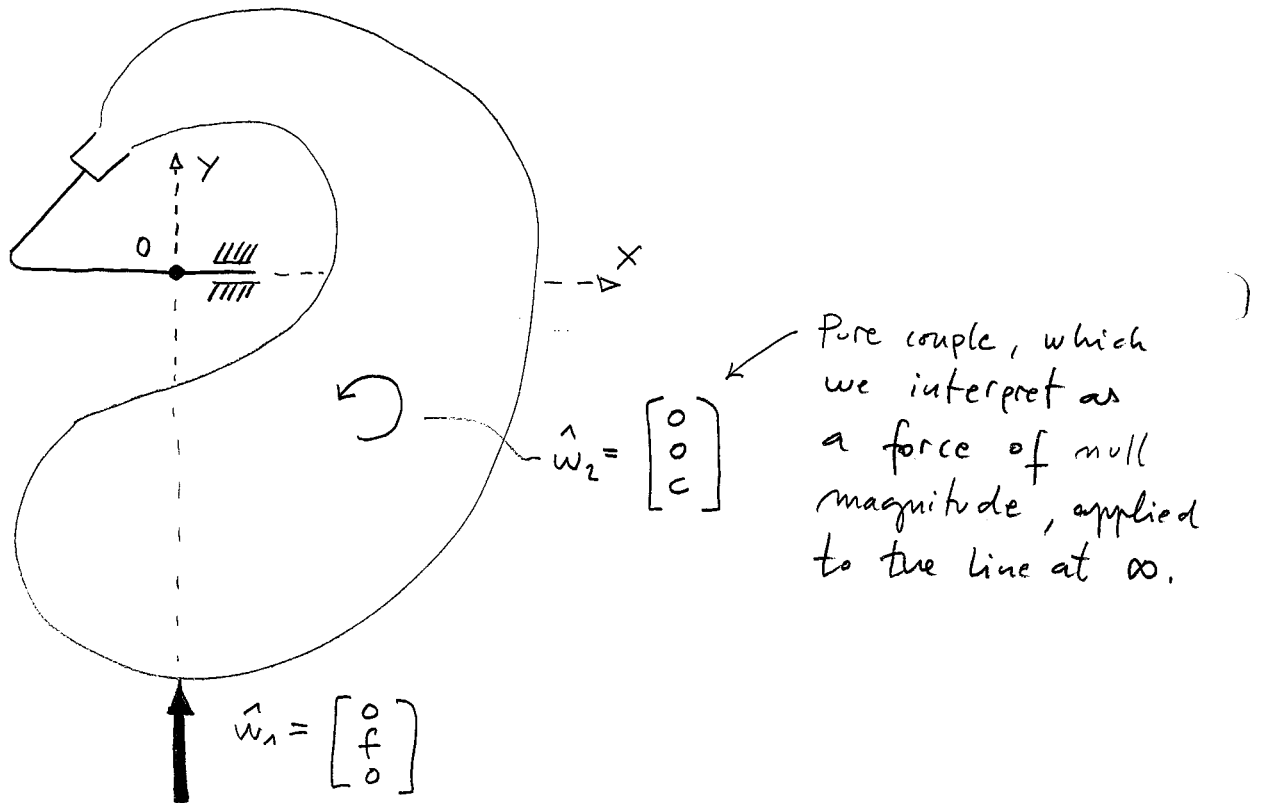


When $f=0$, $\hat{w}_e = \begin{bmatrix} 0 \\ 0 \\ c \end{bmatrix}$ and the line of action is the line at infinity. In this case, \hat{w}_e is a pure couple applied to the end effector

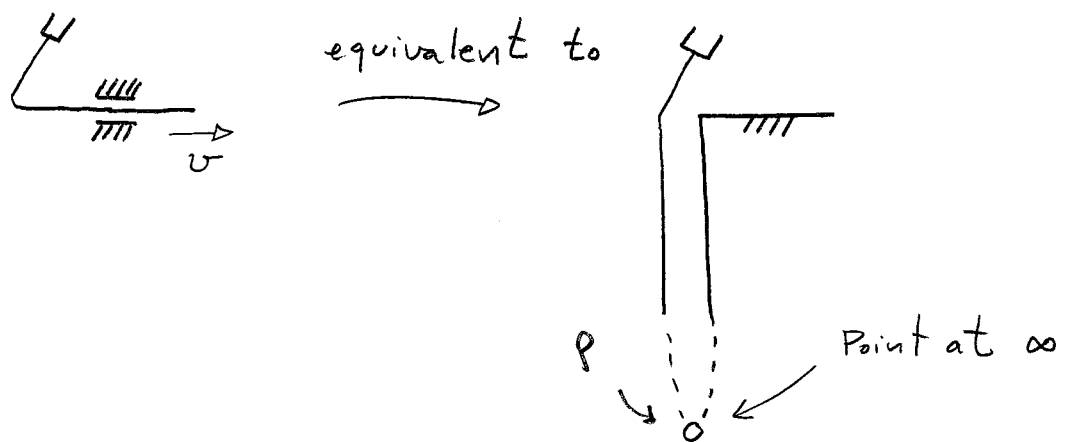
Reciprocity exercise 4

Summarizing, the end effector can structurally support any linear combination of the vertical forces

$$\hat{w}_1 = \begin{bmatrix} 0 \\ f \\ 0 \end{bmatrix} \text{ and pure couples } \hat{w}_2 = \begin{bmatrix} 0 \\ 0 \\ c \end{bmatrix}$$

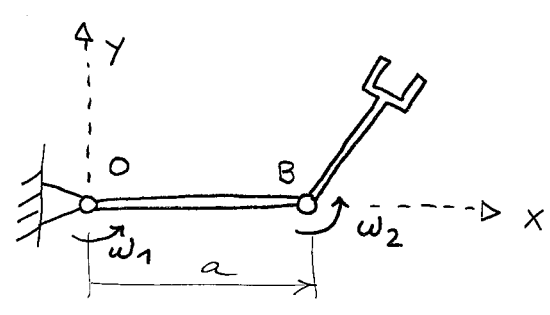


We could have anticipated (from the very beginning) that the structurally supportable forces are those of a line bundle parallel to the y axis, because the prismatic joint is equivalent to a revolute joint at infinity, in the direction orthogonal to the joint velocity:



The end-effector twist, thus, has a line of support r orthogonal to the xy plane at point P . The only lines whose mutual moment with respect to r is null are those that meet P . That is, the lines parallel to the y axis.

Robot 3



The end-effector twists take the form:

$${}^1T_e = \omega_1 \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} + \omega_2 \begin{bmatrix} 0 \\ -a \\ 1 \end{bmatrix}$$

\downarrow $\frac{\text{rad}}{\text{s}}$ \downarrow $\frac{\text{rad}}{\text{s}}$

Thus:

$$\Pi = \left\langle \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}, \begin{bmatrix} 0 \\ -a \\ 1 \end{bmatrix} \right\rangle \quad (\text{in axis coords})$$

W must be the reciprocal complement of Π . Since $\dim(\Pi) = 2 \Rightarrow \dim(W) = 1$. A basis vector for W is the cross product $\begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \times \begin{bmatrix} 0 \\ -a \\ 1 \end{bmatrix} = \begin{bmatrix} a \\ 0 \\ 0 \end{bmatrix}$. Thus:

$$W = \left\langle \begin{bmatrix} a \\ 0 \\ 0 \end{bmatrix} \right\rangle = \left\langle \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \right\rangle \quad (\text{in ray coords})$$

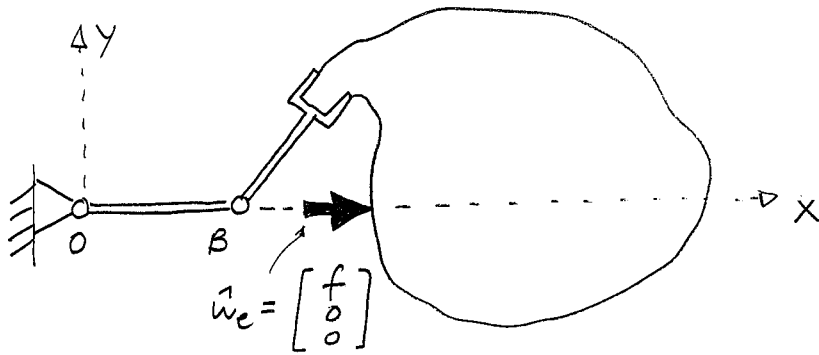
The wrenches of constraint thus take the form

$${}^1W_e = f \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

Newton \leftarrow

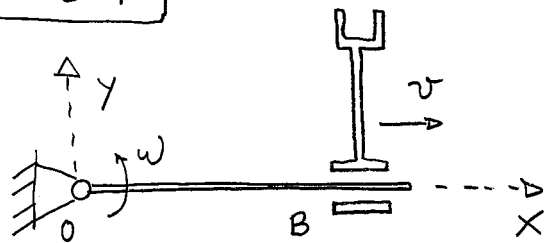
where f is an arbitrary scalar. Thus, the end effector can equilibrate any force on the OB line, without actuating its joints:

Reciprocity exercise 6



We could have anticipated this result by pure geometric reasoning. The only line with a null mutual moment w.r.t. the lines $\begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$ and $\begin{bmatrix} -a \\ 1 \\ 1 \end{bmatrix}$ is line OB.

Robot 4



The end-effector twists take the form

$${}^1T_e = \underbrace{\omega}_{\text{rad/s}} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} + \underbrace{v}_{\text{m/s}} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

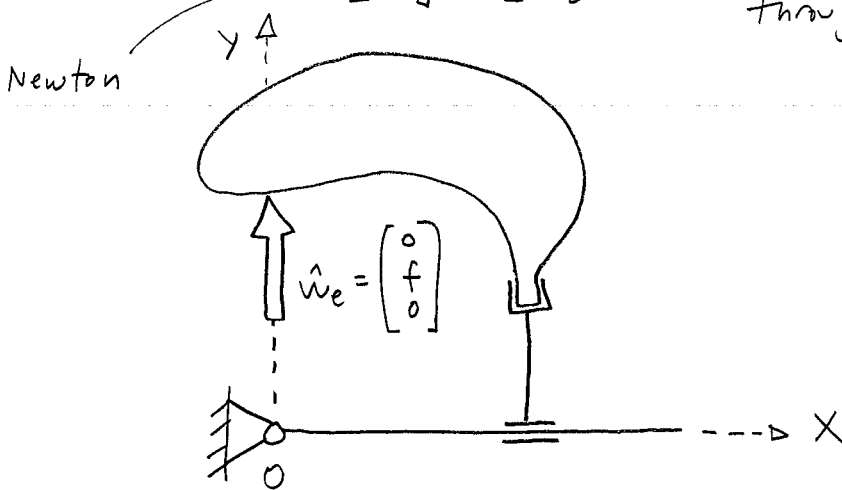
and thus:

$$\Pi = \left\langle \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}, \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \right\rangle \Rightarrow W = \left\langle \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \times \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \right\rangle = \left\langle \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} \right\rangle$$

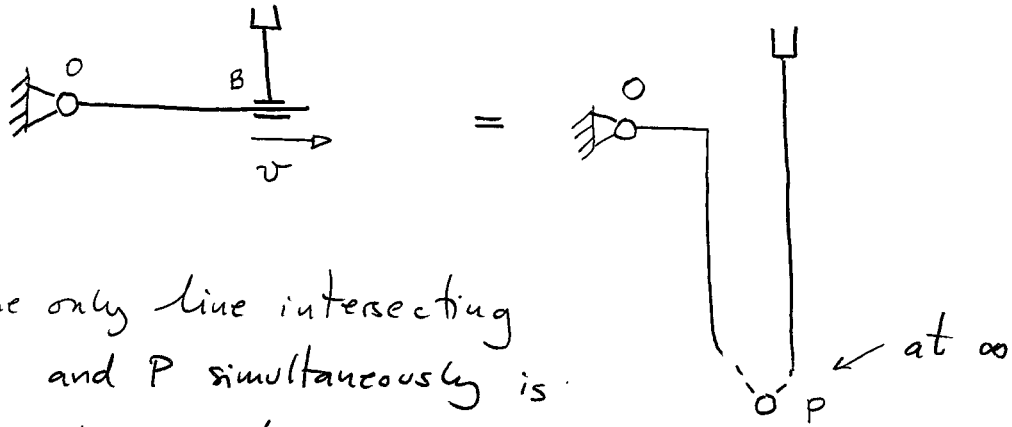
The wrenches of constraint adopt the form:

$$\hat{w}_e = f \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ f \\ 0 \end{bmatrix} \Rightarrow$$

The end effector can structurally support any vertical force through point O



A geometric argument also anticipates this result. The prismatic joint in B is equivalent to a revolute joint at infinity, in the direction orthogonal to the velocity of P:



The only line intersecting O and P simultaneously is the line of the γ axis.